## Abstract

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The low-side pressure of a refrigeration cycle and the refrigerant temperature at the exit of a gas cooler under reference operating conditions are employed as a reference low pressure and a reference refrigerant temperature, respectively, and the high-side pressure of the refrigeration cycle at which the COP of the refrigeration cycle reaches a maximum value under the reference operating conditions is employed as a reference high pressure. In this case, the volume  $v_2$  of a first fluid chamber (72) in the expander (60) just after the closing off of fluid communication from its inlet channel and the volume v<sub>3</sub> of a second fluid chamber (82) in the expander (60) just before the provision of fluid communication with its outlet channel are set to  $v_2=\rho_1v_1r/\rho_2$  and  $v_3=\rho_2v_2/\rho_3$ , respectively, where  $\rho_1$  is the density of saturated gas refrigerant at the reference low pressure,  $\rho_2$  is the density of refrigerant at the reference high pressure and the reference refrigerant temperature, p3 is the density of refrigerant adiabatically expanded from a condition of the reference high pressure and the reference refrigerant temperature into a condition of the reference low pressure, v<sub>1</sub> is the volume of the fluid chamber in the compressor just after the closing off of fluid communication from its suction channel, and r is the rotational speed ratio of the compressor to the expander.